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#### NTP SYSTEM SIMULATION AND DETAILED NUCLEAR ENGINE MODELING

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> NASA Lewis Research Center Plum Brook Station

> > INSPI University of Florida

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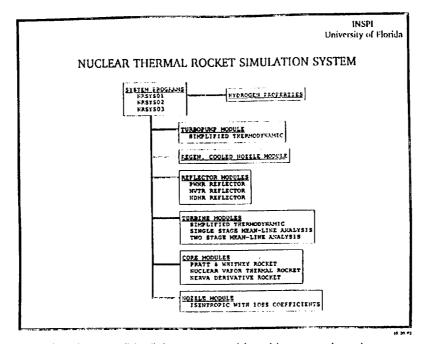
#### MODELING AND ENGINEERING SIMULATION OF NUCLEAR THERMAL ROCKET SYSTEMS

- Modular Thermal Fluid Solver with Neutronic Feedback
- Main Component Modules:

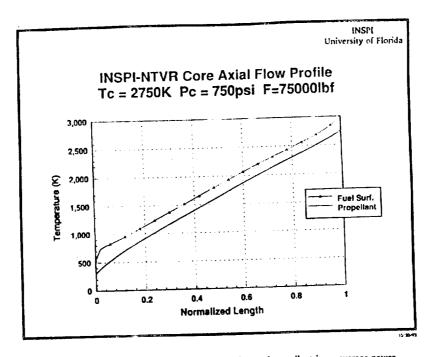
Pipes, Valves, Mixer Nozzle Skirt Pump, Turbine Reflector, Reactor Core

- Hydrogen (Para- and Dissociated) Property Package 10 5 T 5 10,000 K .1 s P s 160 bar
- Models Developed for NTVR, NERVA and XNR 2000
- CFD and Heat Transfer Models for Main NTR Components

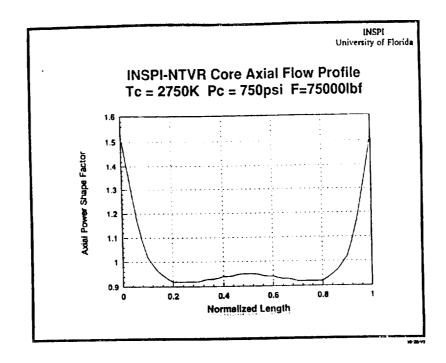
A detailed program for modeling of full system nuclear rocket engines is developed. At present time, the model features the expander cycle. Axial power distribution in the reactor core is calculated using 2- and 3-D neutronics computer codes. A complete hydrogen property model is developed and implemented. Three nuclear rocket systems are analyzed. These systems are: a 75,000 lbf NERVA class engine, a 25,000 lbf cermet fueled engine and INSPI's nuclear thermal vapor rocket.



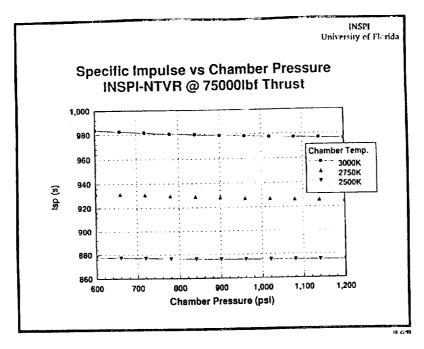
The main program links all the component modules and iterates to arrive at the user specified thrust chamber pressure and temperature and thrust level. Reactor power and propellant flow rate are among outputs of the simulation program. Fuel elements in the core module are prismatic with variable flow area ratio. Fach module divides the relative component into N segments.



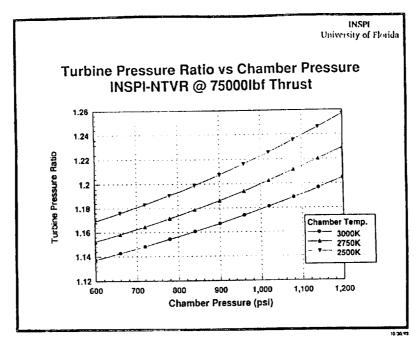
Axial temperature distribution of NVTR fuel surface and propellant in an average power rod. Reactor power is adjusted to achieve the thrust chamber temperature and pressure of 2750 K and 750 psi, respectively.



Normalized axial power distribution in C-C composite fuel matrix NTVR, calculated by DOT-2  $\mathbf{S}_{\mathrm{n}}$  code. The axial power shape factor is an input for the simulation code.

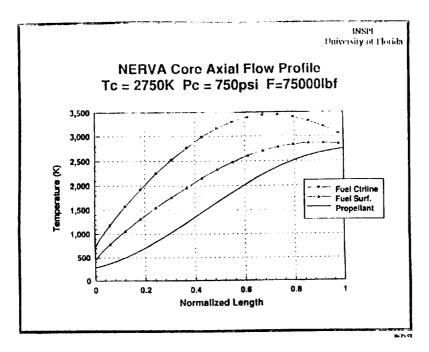


Parametric study of thrust chamber pressure and temperature impact on lsp of NTVR. At higher pressures lsp is less sensitive to thrust chamber temperature.

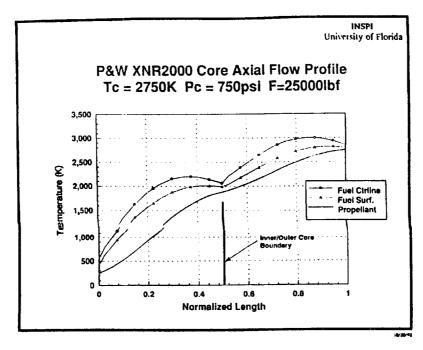


Turbine pressure ratio is sensitive to both thrust chamber pressure and temperature. For thrust chamber pressure of 1200 psi and temperature of 3000 K, the turbine pressure ratio of 1.26 is well within the range of available technology.

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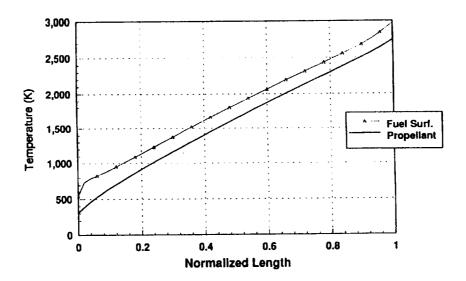


Axial temperature profiles for NERVA-75,000 lbf engine are presented. The maximum fuel temperature is 3490 K at .7 m from the core entrance.

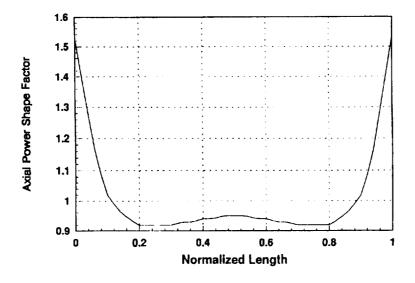


Axial temperature distribution in XNR 2000 core is presented. XNR 2000 features a two path folded flow core fueled with CERMET. The maximum fuel temperature is 3000 K at about 85% from the entrance to the inner core region.

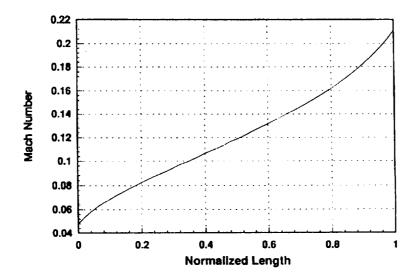
## INSPI-NTVR Core Axial Flow Profile Tc = 2750K Pc = 750psi F=75000lbf



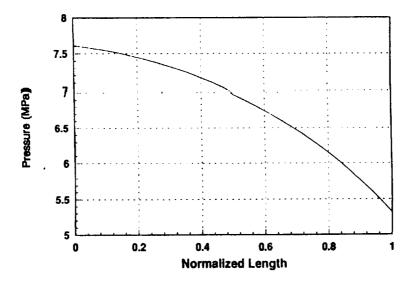
# INSPI-NTVR Core Axial Flow Profile Tc = 2750K Pc = 750psi F=75000lbf



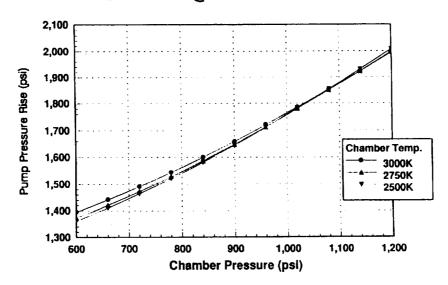
# INSPI-NTVR Core Axial Flow Profile Tc = 2750K Pc = 750psi F=75000lbf



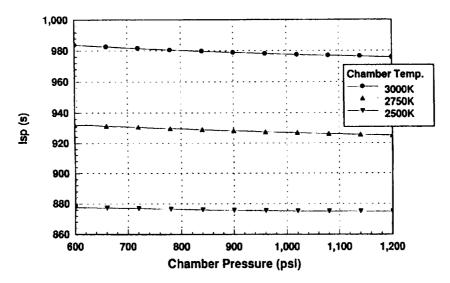
### INSPI-NTVR Core Axial Flow Profile Tc = 2750K Pc = 750psi F=75000lbf



#### Pump Pressure Rise vs Chamber Pressure INSPI-NTVR @ 75000lbf Thrust

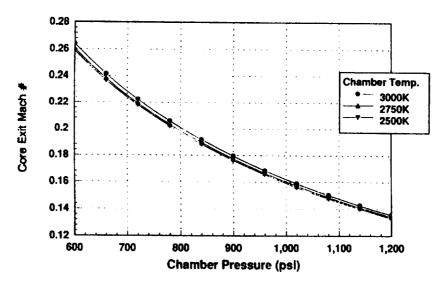


# Specific Impulse vs Chamber Pressure INSPI-NTVR @ 75000lbf Thrust

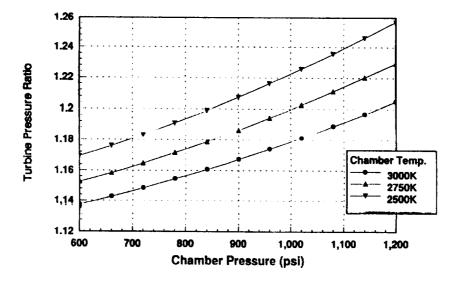


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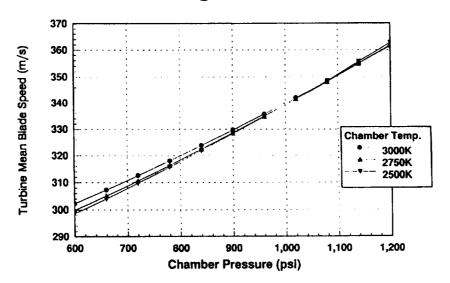
### Core Exit Mach # vs Chamber Pressure INPSI-NTVR @ 75000lbf Thrust



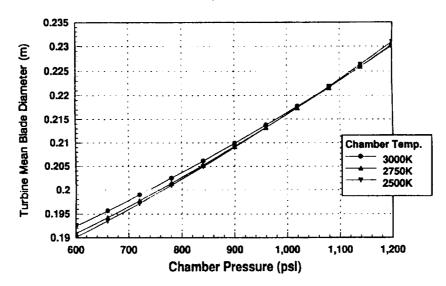
# Turbine Pressure Ratio vs Chamber Pressure INSPI-NTVR @ 75000lbf Thrust



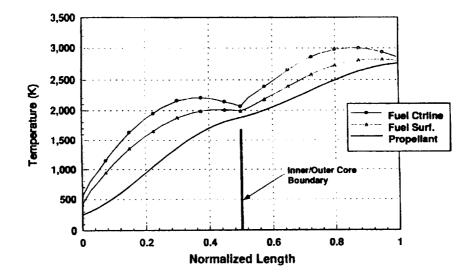
#### Turbine Blade Speed vs Chamber Pressure INSPI-NTVR @ 75000lbf Thrust



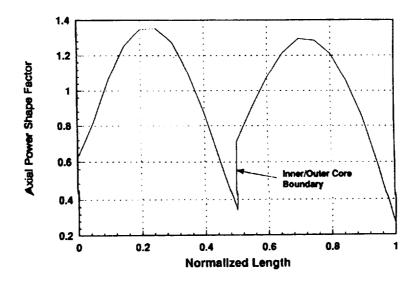
#### Turbine Blade Diameter vs Chamber Pressure INSPI-NTVR @ 75000lbf Thrust



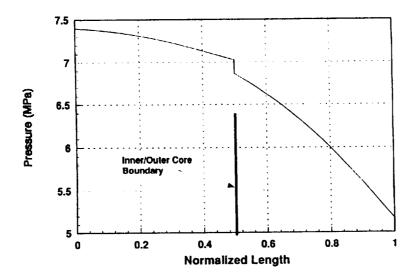
#### P&W XNR2000 Core Axial Flow Profile Tc = 2750K Pc = 750psi F=25000lbf



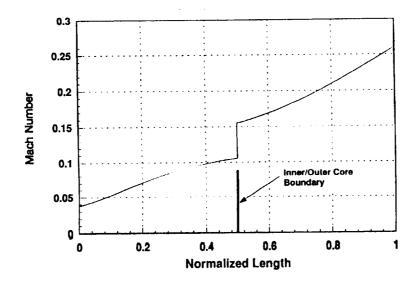
### P&W XNR2000 Core Axial Flow Profile Tc = 2750K Pc = 750psi F=25000lbf



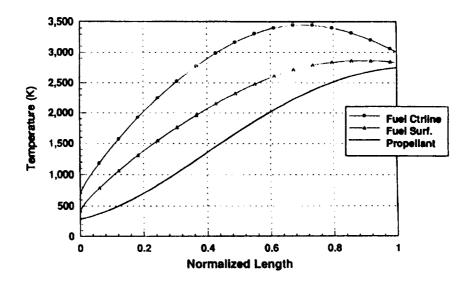
## P&W XNR2000 Core Axial Flow Profile Tc = 2750K Pc = 750psi F=25000lbf



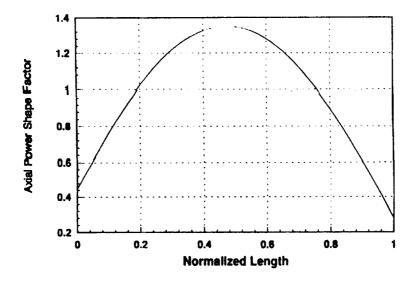
# P&W XNR2000 Core Axial Flow Profile Tc = 2750K Pc = 750psi F=25000lbf



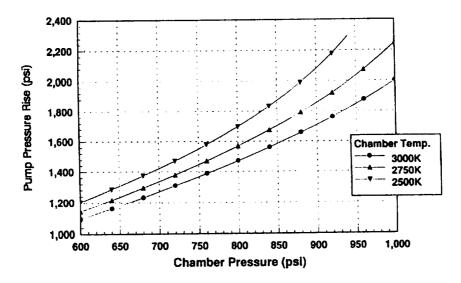
## NERVA Core Axial Flow Profile Tc = 2750K Pc = 750psi F=75000lbf



## NERVA Core Axial Flow Profile Tc = 2750K Pc = 750psi F=75000lbf



#### Pump Pressure Rise vs Chamber Pressure NERVA @ 75000lbf Thrust



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#### EVALUATION OF PARA- AND DISSOCIATED HYDROGEN PROPERTIES AT T = 10 - 10,000 K

NASA/NIST Property Package

 (13.8 < T < 10,000 K and .1 < P < 160 bar)</li>

Molecular Weight, Density Enthalpy, Entropy Specific Heats, Specific Heat Ratio Thermal Conductivity, Viscosity

Hydrogen Property Generator Code Features

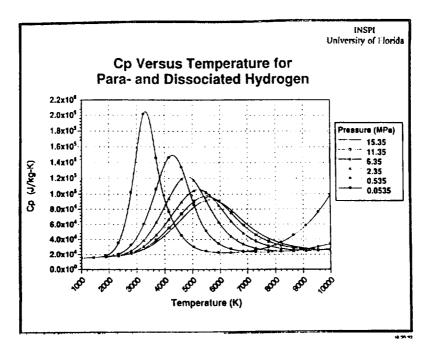
Linear Interpolation Natural Cubic Spline

Least Square Curve Fitting with Pentad Spline Joint Functions

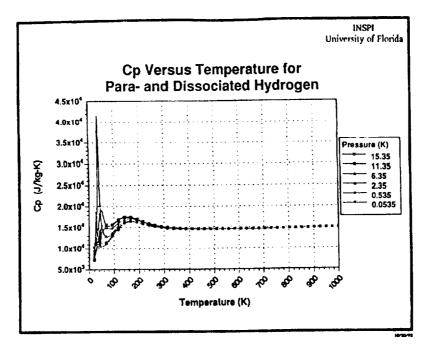
Graphical Representation of Properties

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The hydrogen property generator utilizes two interpolation techniques and a least-square curve fitting routine with a pentad spline function which links least-square fitted pieces together. The property generator package is incorporated into the NTR simulation code and also into a system of CFD-HT codes.

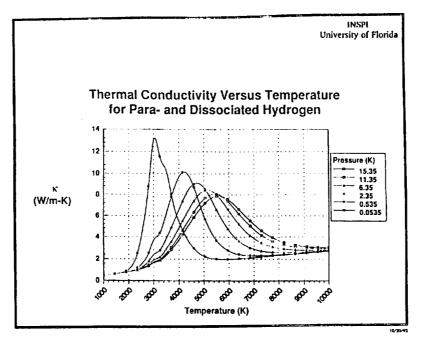


At higher temperatures, the heat capacity data displays smooth behavior. The sharp increase in  $\mathbf{C_p}$  value at temperatures above 2000 K is due to hydrogen dissociation.



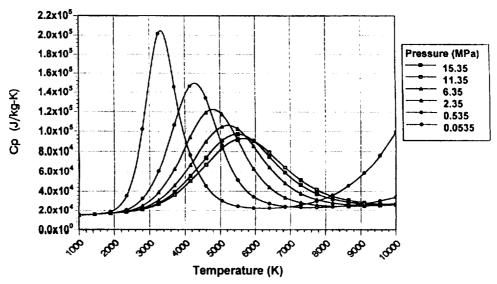
Heat capacity of hydrogen near the critical point shows large gradient and oscillatory behavior. At p=2.35 MPa the property package indicates a sharp peak for  $C_p$ .

NP-TIM-92

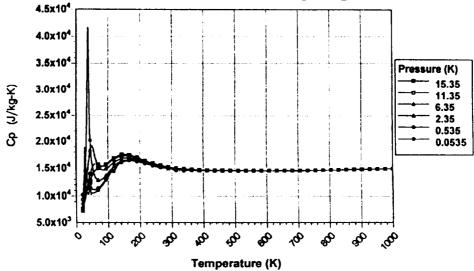


The hydrogen property package is a combination of two subpackages covering the temperature ranges 10 - 3000 K and 3000 - 10,000 K, respectively. The large change of gradients in hydrogen viscosity at 3000 K indicates a non-physical flaw in the model.

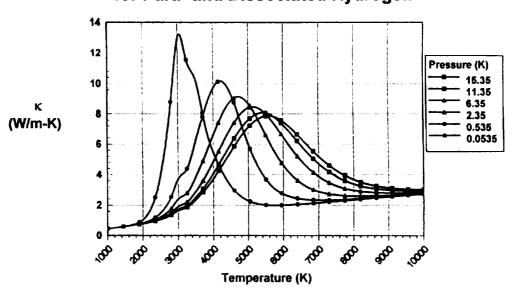
## **Cp Versus Temperature for Para- and Dissociated Hydrogen**



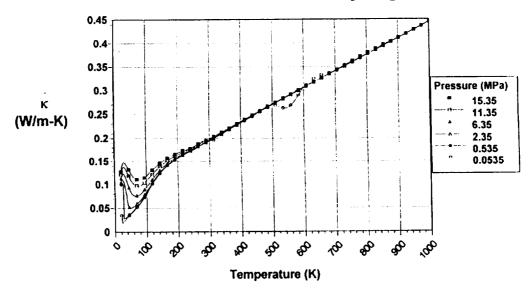




#### Thermal Conductivity Versus Temperature for Para- and Dissociated Hydrogen



## Thermal Conductivity Versus Temperature for Para- and Dissociated Hydrogen

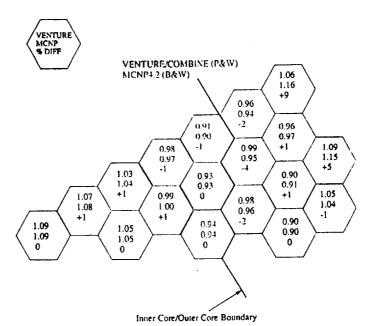


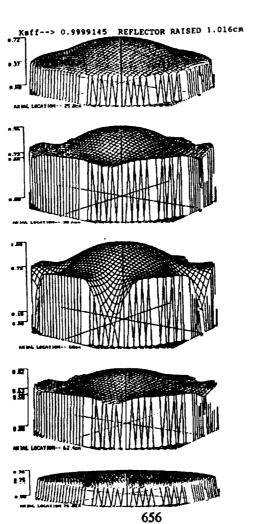
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#### NUCLEAR DESIGN ANALYSIS PACKAGE

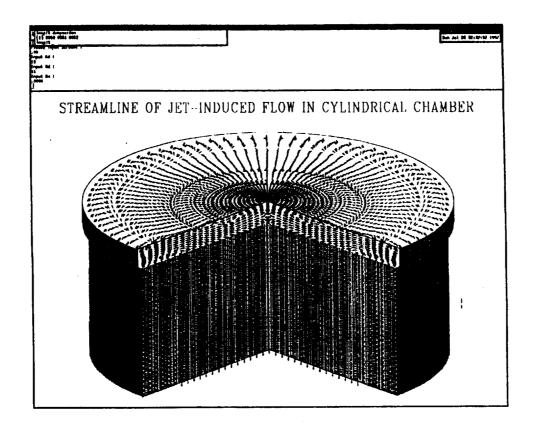
- Multigroup Cross-sections Generated by COMBINE (ENDFB-V)
- MCNP (4.2) for Complex Geometries
- BOLD VENTURE (3-D, Diffusion) for Power Profile and Reactivity Calculations
- ANISN (1-D, S<sub>n</sub>) for Analysis of Heterogeneous Boundaries
- DOT IV (1, 2-D, S<sub>n</sub>) for Analysis of Reflector
- . XSDRNPM (1-D,  $S_n$ ) TWODANT (2-D,  $S_n$ ), NJOY, AMPX for Cross-comparison

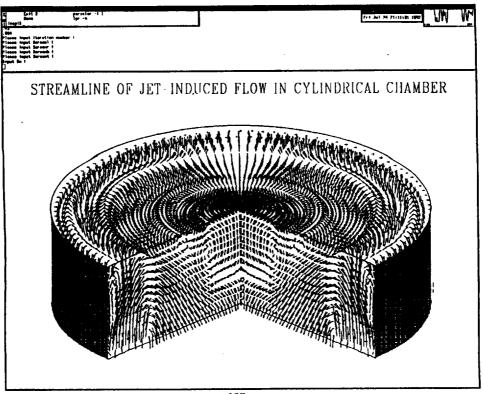
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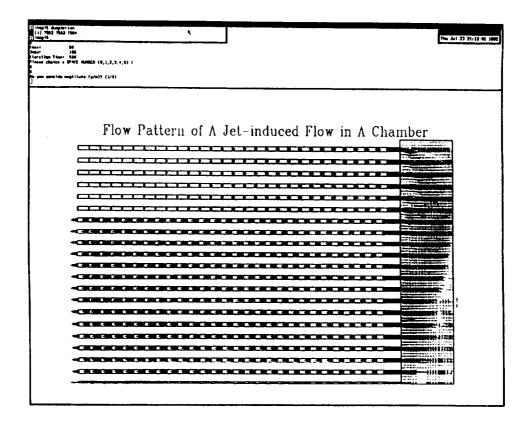


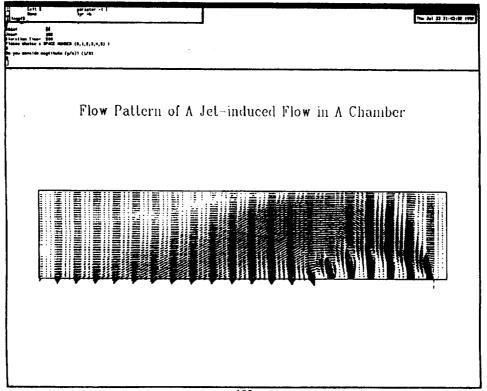


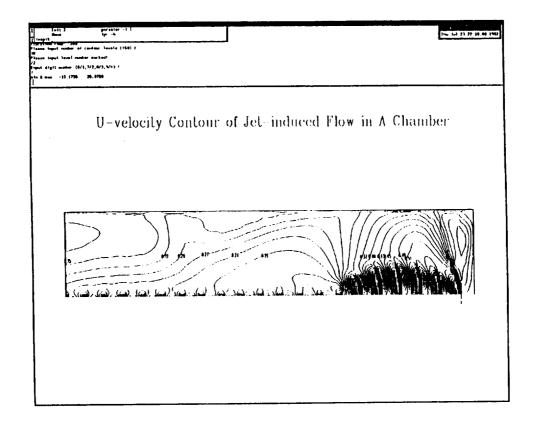
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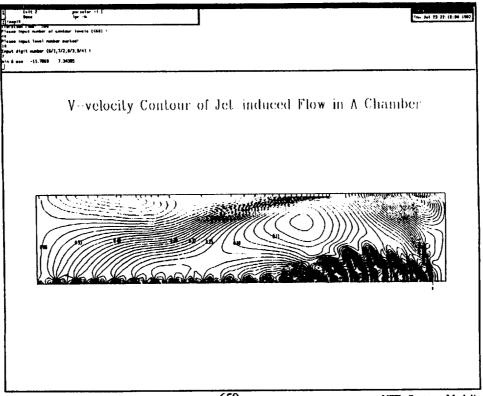






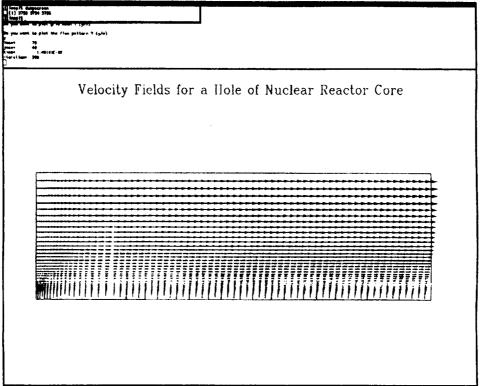






NP-TIM-92 659 NTP: Systems Modeling

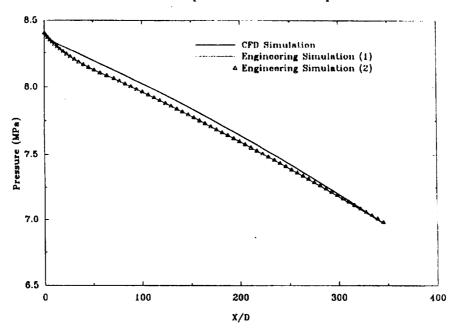
Grid Sy	stem for	a Hole of I	Nuclear Read	ctor Core



NTP: Systems Modeling

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#### Pressure Drop Correlation Comparison



#### Nusselt Number (Nu) and Pressure Drop ( $\Delta P$ ) Correlation Comparison with CFD Analysis

(I) CFD Analysis

Energy Equation

$$\frac{\partial}{\partial X} \left( (e + \sigma_x) u + \tau_{yz} v - K_c \frac{\partial \Gamma}{\partial X} \right) + \frac{\partial}{\partial Y} \left( (e + \sigma_y) v + \tau_{ry} u - K_c \frac{\partial \Gamma}{\partial Y} \right) = S_c$$

Numerical Algorithm: MacCormack hybrid implicit-explicit, finite volume method

Conductive Heat Flux

$$q_{\epsilon}^{"} = -K_{\epsilon} \left[ \frac{\partial T}{\partial r} \right]_{r=R}$$

Convective Heat Flux

$$q_x = h_c(T_w - T_b)$$

$$\int \rho C_p u T dA$$

$$T_b = \frac{A}{\int \rho C_p u dA}$$

Convective Heat Transfer Coefficit

$$h_c = -\frac{K_c \left[\frac{\partial T}{\partial \tau}\right]_{\tau=R}}{T_w - T_b}$$

Nusselt Number

$$Nu = \frac{h_c D}{K_c}$$

#### DIFFUSION APPROXIMATION

$$g_t'' = -\frac{4}{3a_g} \nabla e_b = -\frac{16\sigma_{tg} T^3}{3a_g} \nabla T = -K_t \nabla T$$

$$K_t = \frac{16\sigma_{tg} T_v^3}{3a_g}$$

Using the perfect gas law,

$$k_r = \frac{16\sigma_{sp}\kappa}{3\sigma_{pb}p}T^4$$

WHERE

ag : Rosseland Mean Opacity

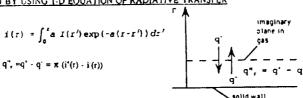
osa : Stefan-Boltzmann Constant

σ<sub>ph</sub>: Photon Collision Cross Section per Molecule κ : Boltzmann's Constant

P : Gas Pressure

T : Gas Temperature

#### APPROXIMATION BY USING 1-D EQUATION OF RADIATIVE TRANSFER



WHERE

i'(t): Radiation Intensity in the Positive Direction( From Gas to Boundary) i(r): Radiation Intensity in the Negative Direction( From Boundary to Gas)

I(r) : Source Function (=σT<sup>4</sup>/π)

#### Nusselt number & Prandtl number

$$Re = \frac{\rho uD}{\mu(T_b, T_1)}$$

$$Pr = \frac{\mu(T_b)C_p(T_b)}{K_c(T_b)}$$

(III) Pressure Drop

#### Compressible Flow

$$\Delta P = \frac{RG^2 T_m}{P_m} \left( \ln \frac{\rho_1}{\rho_2} + \frac{2f\Delta Z}{D} \right)$$

$$R = \frac{C_P(\gamma - 1)}{\gamma}$$

$$G = \rho_1 \left( \frac{V_1 + V_2}{2} \right)$$

$$T_m = \frac{T_1 + T_2}{2}$$

$$P_m = \frac{P_1 + P_2}{2}$$

#### Incompressible Flow

$$\begin{split} \Delta P &= 2f \frac{\Delta Z}{D} \rho_1 V_1^2 \left( \frac{T_1 + T_2}{2T_1} \right) + \rho_1 V_1^2 \left( \frac{F_2}{F_1} - 1 \right) \\ f &= 0.0014 + \frac{1}{5} Re^{-0.32} \end{split}$$

- (II) Nusselt Number Correlations
- (1) Colburn Equation

$$Nu = 0.023 Re^{0.8} Pr^{\frac{1}{2}}$$

(2) Dittus-Boelter Equation

$$Nu=0.023Re^{0.8}Pr^{0.3}$$

(3) Sieder-Tate Equation

$$Nu = 0.027 Re^{0.8} Pr^{\frac{1}{3}} \left( \frac{\mu_b}{\mu_w} \right)^{0.14}$$

(4) Petukov Equation

$$Nu = \frac{RePr}{X} \left(\frac{f}{2}\right)$$

$$X = 1.07 + 12.7 \left(Pr^{\frac{2}{3}} - 1\right) \left(\frac{f}{2}\right)^{\frac{3}{3}}$$

$$f = 0.0014 + \frac{1}{8}Re^{-0.32}$$
(5) Karmon-Boelter-Martinelli Equation

$$N_{s} = \frac{RePr\sqrt{\frac{I}{2}}}{0.833\left(5Pr + 5ln(5Pr + 1) + 2.5ln\left(Re\frac{\sqrt{\frac{I}{2}}}{60}\right)\right)}$$
$$f = 0.0014 + \frac{1}{8}Re^{-0.32}$$

Axial Distance Correction

$$Nu(x) = Nu\left(1.957\left(1 + \frac{x}{D}\right)^{-0.15}\right)\sqrt{\frac{T_b}{T_w}}$$

$$Nu(x) = Nu\left(1 + \frac{2\ln\frac{P_T}{DT}}{\frac{1}{D}}\right)$$

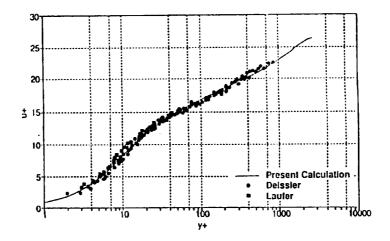


Figure 6.2 Velocity distribution for a fully developed turbulent flow in tube. (Re=1.6 E+4)

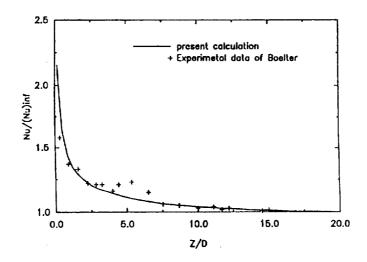


Figure 6.17 Musselt number vs. axial position for a developing isothermal pipe flow at a Reynolds number of 53000.  ${\rm (Nu)_{inf}}$  is the Nusselt number evaluated at Z/D=20.

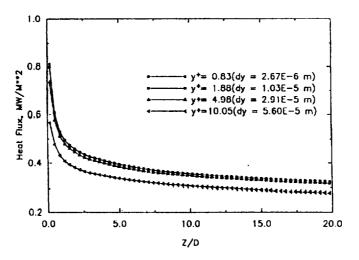


Figure 6.15 Heat transfer rates obtained by Navier-Stokes solver for various boundary cell size. A 60×60 grid is used.  $(T_{1s}=4000\ K,\ T_{w}=1800\ K,\ P_{1s}=1\ atm,\ and\ P_{aux}=0.5\ atm)$ 

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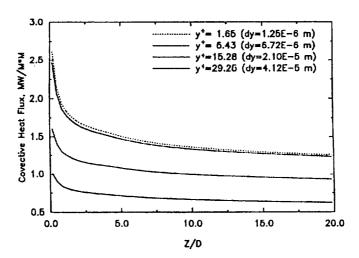


Figure 6.16 Heat transfer rates obtained by Navier-Stokes solver for various boundary cell size. A 60x80 grid is used. ( $T_{in}$ =4000 K,  $T_{ir}$ =1800 K,  $P_{ir}$ =10 atm, and  $P_{out}$ =9.5 atm)

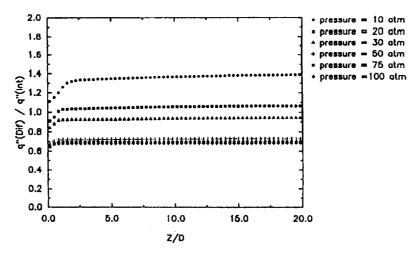


Figure 6.19 Comparative result between diffusion approximation and 1-D integral approximation for varying the gas opacity due to different flow conditions.

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